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Compression-softening effect in prestressed beams: Experimental-finite-element vibration analysis of post-tensioned thin-walled steel-box-girders

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In published works, there is a discrepancy about the effect of pre and post-tensioning forces on the beam 8 9 dynamics. Particularly, the dynamic effects caused by an external compressive force and that due to post-10 tensioning appear to be quite different. To solve this conflict, free transverse vibrations of a simply 11 supported post-tensioned thin-walled steel-box-girder were investigated. Its fundamental frequencies 12 were then measured for different values of post-tensioning force. Subsequently, the experimental data 13 were compared with a formula taking shear effects into account. The experimental data were also 14 compared with two high-fidelity finite-element models. According to the findings of this study, and 15 comparisons with several tests reported in the literature, the beam dynamics due to a compressive force 16 and that caused by post-tensioning is different only when the cables are in contact with the surrounding 17 beam's section. Depending on this feature, the dynamics of pre and post-tensioned beams is ruled by 18 compression-softening effect. Moreover, in thin-walled steel-girder-bridges draped with a deviator, the 19 fundamental frequency as indicator for post-tensioning loss identification is made doubtful.

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Keywords: Finite-element model; Frequency; Post-tensioning; Shear deformation; Straight cable; Thin walled box-girder

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24 **1. Introduction**

Prestressing techniques were widely used for realizing important structures and infrastructures owing to their superior performance and high durability. The prediction of change in natural vibration frequencies with varying prestressing force in beams is a crucial problem, see Bai-jian et al. [1], Gan et al. [2] and references cited therein. Structural engineers should be able to estimate changes in natural frequency of prestressed structures over the course of their design life to ensure their safety. This argument is significant in the field of bridges and wind turbine towers, both of which are susceptible structures to dynamic

1 excitation [3, 4]. E.g., vibration measurements, taken during the operation stage of a concrete bridge or a 2 prestressed steel wind tower, are useful applications to control their calculation model, evaluate their 3 stiffness or monitor their internal stresses [5, 6]. Bonopera et al. [7] fully illustrated a state-of-the-art 4 review of researches conducted worldwide about this topic. Particularly, Hamed and Frostig [8], Jaiswal 5 [9], Limongelli et al. [10] and Bonopera et al. [11, 12] declared that the natural frequencies and stiffness 6 of Prestressed Concrete (PC) girder-bridges, with a parabolic or a straight tendon, only significantly vary 7 under the effect of crack initiation or crack re-opening. Pisani et al. [13] confirmed that, for prestressed 8 beams, modal parameters provide information regarding damage only when the members are close to the 9 ultimate conditions. Previously, Noble et al. [14, 15] executed several experiments on a number of post-10 tensioned steel and concrete beams in small-scale. The researchers declared that the small reduction in 11 fundamental frequency with increasing post-tensioning force is not related to the softening effect. Thus, 12 they deduced that the compression-softening theory must be eliminated from discussion of all forms of 13 pre and post-tensioned structures. The researchers also claimed that the dynamic effect of a compressive 14 force and that of a post-tensioning on a beam are different on a phenomenological level. Consequently, 15 they retained that a prestressing force should always be considered as an axial force internally applied. 16 Gan et al. [2] sustained that this divergence in Noble et al. [15] was caused by the closure effect of 17 shrinkage cracks and/or microcracks inside the post-tensioned concrete beams which, in turn, was not 18 considered in the theoretical predictions of fundamental frequency. According to the literature above 19 mentioned, there is a meaningful disagreement among researchers about the effect of pre and post-20 tensioning forces on the beam dynamics. Therefore, it was deemed that the results regarding this topic 21 need additional experimental investigations on steel beams rather than those in reinforced concrete, with 22 the aim to prevent stiffening effects caused by the microcrack closure [9] and by the time-increment of 23 Young's modulus regarding the consolidation/hardening of concrete [11, 12, 16]. Such investigations had 24 to be conducted on post-tensioned beams in a larger scale for avoiding that any mass at the end constraints, 25 usually composed of load cells and/or hydraulic jacks, was greater than the self-mass of the beams 26 themselves. In fact, it is what occurred in Noble et al. [14] where the fundamental frequencies were 27 affected by such masses [11, 12]. Accordingly, the conclusions of Noble et al. [14] required to be revised.

Hence, this work was mainly necessary for properly clarifying and evaluating the dynamics of prestressed beams. Indeed, a correct evaluation of fundamental frequency is crucial for designing new bridges and prestressed wind turbine towers, monitoring and determining the conditions of existing ones and related components. Furthermore, the reproduction of experimental data using correct algorithms enables effective interpretations of in situ measurements and simulations of virtual laboratory tests [17].

6 This work was a research sub-program developed at the National Center for Research on 7 Earthquake Engineering (NCREE) according to a campaign study on girder-bridges which initiated in 8 2015 by testing PC beam specimens [11, 12, 16–20]. Given the conflicts in the aforementioned findings, 9 firstly, free transverse vibrations were induced to a simply supported post-tensioned thin-walled steel-10 box-girder in larger scale and with eccentric straight cables. Different post-tensioning forces were applied. 11 The specimen was characterized by initial second-order curvatures, due to the post-tensioning force 12 causing two equal bending moments at the end constraints, and an axial end constraint, due to the stiffness 13 of the cables [21, 22]. Secondly, the experiments executed by Noble et al. [14] were analyzed. Indeed, the 14 researchers similarly investigated the compression-softening theory in two simply supported thin-walled 15 steel-box-members post-tensioned by a straight cable. The two specimens in small scale had different 16 slenderness. Subsequently, the fundamental frequencies measured from testing, and those obtained by 17 Noble et al. [14] 's experiments, were compared with a formula based on the Timoshenko shear beam 18 model [23], where the post-tensioning force is equivalent to a compressive force externally applied. The 19 frequencies were also compared with two high-fidelity Finite-Element (FE) models including shear deformation, in which the post-tensioning force is applied as initial tension in the cables. In the FE 20 21 analyses, the contact between cables and surrounding sections was additionally simulated. According to 22 the obtained findings, the dynamic effect of a compressive force and that of a post-tensioning are 23 phenomenologically different only when the cables are in contact with the surrounding beam's section. 24 Depending on this feature, the dynamics of all forms of pre and post-tensioned beams is ruled by 25 compression-softening effect. Similarly to what deduced for PC girder-bridges [7, 19, 20], the 26 fundamental frequency as parameter for post-tensioning loss identification is made doubtful in thin-27 walled steel-girder-bridges draped with a deviator.

1 **2. Reference solutions**

2 The reference solution, proposed in this study, describes the prestressed beam dynamics based on 3 Timoshenko beam model, where transverse shear deformability is considered [23, 24]. The shear beam 4 theory is often used to model the behavior of structures both for stability or dynamic analyses [25–27]. 5 The effect of rotational inertia is neglected because it may be appreciable only at high frequencies [28]. 6 Particularly, the vibrational response of an axially unloaded simply supported beam takes first-order 7 effects into account including shear deformation [29]. Vice versa, an externally compressed member of 8 length L allows to consider the second-order effects in a beam pre or post-tensioned by a straight tendon 9 which can be or not in contact with the surrounding section (Fig. 1). This configuration characterized the 10 experimental investigations illustrated in Section 3.1. Indeed, a thin-walled steel-box-girder was subjected 11 to different post-tensioning forces ($N_{0x,aver}$) exerted by eccentric straight cables which, in turn, were free 12 to vibrate inside the Rectangular Hollow Section (RHS) [30] (Sections 3.2 and 3.3). In short, the cables 13 were only in contact at the girder ends. Their cross sectional area were assumed as unchanged after 14 deformations. Therefore, the eccentricity (e) of post-tensioning forces ($N_{0x,aver}$) was theoretically zero. The flexural shape $v^{(0)}$ of the girder's fundamental frequency ($f_{\text{shear},1,II}$), with total self-mass per unit length 15 16 m_{tot} , is represented in Fig. 1, according to the shear beam model which, in turn, included the second-order 17 effects.



Fig. 1. Reference solution. Prismatic beam pre or post–tensioned by eccentric straight tendon. Fundamental flexural shape $v^{(0)}$ according to the shear model. The dashed line represents the undeformed configuration.

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When the member in Fig. 1 meets all the requirements of Euler-Bernoulli theory, it is subjected to small displacements and rotations. Thus, the second-order *n*th natural frequency ($f_{\text{E-B.},n,II}$) of a simply supported post–tensioned steel-girder with a straight unbonded tendon is described by the following equation where, the effects of longitudinal vibrations and warping are not considered [23, 29]:

$$f_{\text{E-B},n,II} = \frac{1}{2\pi} \sqrt{\frac{\alpha^2 n^4 \pi^4}{L^4}} \sqrt{1 - \frac{N_{0x,\text{aver}}}{N_{\text{crE},n}}} \cdot \qquad n = 1, 2, \dots$$
(1)

1 Here the coefficient $\alpha^2 = EIg/m_{tot}$ contains Young's modulus E and cross sectional second moment of the 2 3 area I of the thin-walled box-girder. The gravitational acceleration g = 9.81 m/s². Instead, the second square root contains the *n*th critical buckling load of the simply supported girder given by $N_{\text{crE},n} = n^2 \pi^2 EI$ 4 L^2 . The self-mass per unit length of the straight unbonded tendon (m_{tendon}) does not affect the beam 5 dynamics. Likewise, no stiffening effect is induced by the cables under post-tensioning ($N_{0x,aver}$). 6 7 Assuming n = 1 in Eq. (1), the second-order fundamental frequency $(f_{E-B,1,II})$ is obtained. In detail, Eq. 8 (1) reduces to Eq. (2) reported in Bonopera et al. [11], where the *n*th critical buckling load ($N_{crE,n}$) becomes 9 equal to the Euler buckling load $N_{crE,1} = \pi^2 EI/L^2$. Furthermore, disregarding the term containing the post– 10 tensioning force $N_{0x,aver}$, Eq. (3), likewise reported in Bonopera et al. [11], is achieved. Such formula 11 represents the first-order fundamental frequency $(f_{E-B,1,J})$ of simply supported PC girder-bridges with a 12 straight bonded or unbonded tendon. Indeed, the dynamic effect of a pre or post-tensioning force ($N_{0x,aver}$) 13 is negligible in these types of PC girder-bridges [11]. Conversely, when the aforementioned thin-walled 14 girder meets all the requirements of the shear beam model, its first-order *n*th natural frequency ($f_{\text{shear},n,l}$) is expressed by the following equation which, in turn, does not consider longitudinal vibrations, warping 15 16 and rotary inertia [29]:

$$f_{\text{shear},n,I} = \frac{1}{2\pi} \sqrt{\frac{\alpha^2 n^4 \pi^4}{L^4 \left(1 + \frac{n^2 \pi^2 r^2}{L^2} \frac{E}{k G}\right)}} . \qquad n = 1, 2, \dots$$
(2)

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In Eq. (2), the coefficient $r^2 = I/A$, where A is the cross sectional area of the thin-walled girder. Elastic shear modulus G = E/[2(1 + v)]. v is Poisson's ratio. k is instead the shear coefficient for generic thinwalled hollow sections which is described by the following equation [31]:

$$k = \frac{10(1+\nu)(1+3m)^2}{(12+72m+150m^2+90m^3)+\nu(11+66m+135m^2+90m^3)+10n^2[(3+\nu)m+3m^2]}.$$
(3)

Here we have the parameters $m = b t_1 / h t$ and n = b / h, where internal width *b*, thickness of the flanges t_1 , height *h*, and thickness of the web *t*. According to the above, based on the shear model, and when secondorder effects are assumed, the *n*th natural frequency ($f_{\text{shear},n,II}$) of a simply supported post-tensioned thinwalled steel-girder with a straight unbonded tendon (Fig. 1) can be expressed as:

$$f_{\text{shear},n,II} = \frac{1}{2\pi} \sqrt{\frac{\alpha^2 n^4 \pi^4}{L^4 \left(1 + \frac{n^2 \pi^2 r^2}{L^2} \frac{E}{k G}\right)}} \sqrt{1 - \frac{N_{0x,\text{aver}}}{N_{\text{crE},\text{shear},n}}} . \qquad n = 1, 2, \dots$$
(4)

2 In Eq. (4), the second square root, originating from Eq. (1), contains the *n*th critical buckling load with 3 the effect of shear deformation ($N_{crE,shear,n}$) which is given by the following approximate formula [23]:

$$N_{\rm crE, shear, n} = \frac{N_{\rm crE, n}}{1 + \left(N_{\rm crE, n}/GA_0\right)}.$$
 $n = 1, 2, ...$ (5)

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5 In Eq. (5), the *n*th critical buckling load of a simply supported girder $N_{crE,n}$, whereas the parameter $A_0 = A$ 6 */m. m* is the correction coefficient which takes into account the nonuniform distribution of the shear 7 stresses throughout the cross sectional area (*A*). For a generic thin-walled section, which bends in the 8 plane of the web, $m \simeq A/A_w$. A_w is the cross sectional area of the web. Substituting $N_{crE,n} = n^2 \pi^2 EI/L^2$ for 9 the denominator, and the expression $I = r^2 A$ then, Eq. (5) can be reformulated by Eq. (6) which is reported 10 as follows [23]:

$$N_{\text{crE,shear},n} = \frac{N_{\text{crE},n}}{1 + n^2 m \left[\frac{\pi}{(L/r)} \right]^2 (E/G)} \cdot \qquad n = 1, 2, \dots$$
(6)

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We see that the smallest critical buckling load ($N_{crE,shear,n}$) occurs for n = 1. When slenderness (L/r) is sufficiently large, shear effects are negligible, and the classical Euler solution is recovered. From Eq. (6) we observe that shear strains decrease the critical load ($N_{crE,shear,n}$). Therefore, the shear correction becomes significant for short post-tensioned thin-walled girders (small L/r) made with a high strength steel, so that the short beam still fails due to buckling rather than yield [23].

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18 **3. Experimental investigation**

19 3.1. Thin-walled steel-box-girder post-tensioned by eccentric straight cables

Given the discrepancies created about the effect of pre and post-tensioning forces on the beam dynamics (Section 1), a thin-walled RHS member of b = 150 mm, h = 350 mm, $t = t_1 = 9$ mm, and made with a high strength steel, was adopted (Fig. 2). Steel was chosen to prevent any stiffening effects caused by the crack and/or microcrack closure [2, 9] and by the time-increment of Young's modulus regarding the consolidation/hardening of concrete [11, 12, 16]. The member was subjected to a series of post-tensioning forces exerted by a straight unbounded tendon with a small eccentricity (e/h = 0.23) with respect to the

1 RHS centroid (Fig. 2). The tendon was composed of 5 steel "seven-wire" strands, of 15.2 mm in diameter, 2 which passed internally through the RHS body and only anchored at its ends (Fig. 2). The tendon's self-3 mass per unit length ($m_{\text{tendon}} = \rho_{\text{tendon}} \times A_{\text{tendon}}$) was 0.0533 kN/m. Two steel supports were arranged at the RHS ends to create pinned-end restraints, resulting in a clear span L (Fig. 2). $132 \times 176 \times 6$ mm³ vertical 4 5 stiffeners, made with the same structural steel, were welded into the RHS upper part at a distance of ≈ 1.0 6 m to prevent instability modes with respect to the girder's vertical axis under post-tensioning (Fig. 2). 7 Furthermore, the RHS ends were covered by a $250 \times 450 \times 20$ mm³ steel plate, likewise made with the same 8 steel. 12 threaded bars, of 16 mm in diameter, were utilized for the fastening, for a total mass at the RHS 9 ends of 317.1 N [Figs. 3(a)–(b)]. All corresponding material and geometric properties of the thin-walled 10 RHS member were listed in Table 1. Notably, unit weight, Young's modulus and yielding stress of high 11 strength steel were evaluated by tensile tests considering the average values obtained from 4 hourglass 12 specimens [32].

RHS $b = 150 \text{ mm} \times h = 350 \text{ mm} \times t = t_{I} = 9.0 \text{ mm}$





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Property	RHS $50 \times 30 \times 3 \text{ mm}^3$
$I (\mathrm{mm}^4)$	1.33399×10^{8}
$A (\text{mm}^2)$	8,676
$A_w (\mathrm{mm}^2)$	5,976
Slenderness	56
<i>k</i> [Eq. (3)]	0.687
<i>L</i> (m)	6.98
ρ_{beam} (kN/m ³)	77.01
E (GPa)	205.12
f_{yk} (MPa)	390
G (GPa)	82.05
υ	0.25
$A_{\text{tendon}} (\text{mm}^2)$	695
$\rho_{tendon} (kN/m^3)$	76.65
Etendon (GPa)	200
σ_{uy} (MPa)	1,860
<i>e</i> (mm)	80

Table 1. Properties of the thin-walled steel RHS member (Fig. 2).



Fig. 3. (a) Laboratory test rig. (b) Load cell and circular plate at one RHS end. Reference seismometer (Af) to the floor (Fig. 4). (c) Seismometer A1 on the top of quarter cross section (Fig. 4).

2 3.2. Experimental set-up

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3 The thin-walled steel-box-girder was located in a test rig [Fig. 3(a)]. At one end constraint, a hydraulic 4 jack, of 1,000 kN in force capacity, was utilized to apply the post-tensioning forces by pulling the cables 5 outward (Fig. 2). Yet, a load cell, of 130 mm in diameter, 120 mm in length, 1,000 kN in force capacity, 6 2 mV/V in sensitivity, and 78.5 N in mass, was fixed at either RHS ends to measure the applied post-7 tensioning forces N_{0x1} and N_{0x2} [Fig. 2 and Fig. 3(b)]. A circular steel plate, of 130 mm in diameter, 60 8 mm in length, and 49.4 N in mass, was additionally located between the load cells and the end of cables 9 [Fig. 3(b)]. 6 mean post-tensioning forces ($N_{0x,aver}$) were totally applied in values of $\approx 117, 214, 332, 421,$ 10 501 and 598 kN, to respectively induce second-order effects of 2.2, 4.0, 6.4, 8.2, 9.9 and 12.1% of the 11 girder's critical buckling load $N_{crE,1}$ or $N_{crE,shear,1} = 5,543$ kN. Notably, the maximum tension force in each 12 cable was not to exceed the value of \approx 140 kN, according to the laboratory's safety conditions. Thus, the 13 maximum tensile strength reached in the tendon, $\sigma_{\text{tendon,max}} = N_{0x,\text{aver,max}} / A_{\text{tendon}} = 598 \text{ kN} / 695 \text{ mm}^2 = 860$ 14 MPa, was $\approx 46\%$ of yielding stress σ_{uy} (Table 1). The different post-tensioning forces (N_{0x1} and N_{0x2}) 15 measured at the RHS ends occurred because of the post-tensioning losses due to anchorage slips and 16 relaxation of cables (Fig. 2). The measuring system included 5 seismometers installed along the RHS 17 body (Af, A0, A1, A2 and A3) as depicted in Fig. 4. Particularly, 5 high-precision servo velocity 18 seismometers, lightweight (2.65 N), of 50 mm in diameter, 70 mm in length, and 5 mV/gal in sensitivity, 19 were selected (VSE-15D, Tokyo Sokushin). 2 seismometers of which, marked as A1 and A2, were 20 vertically fixed on the top of quarter and midspan cross sections, i.e., at i = 1 and 2, with the aim to 21 respectively collect acceleration data with respect to the RHS strong axis [Fig. 3(c) and Fig. 5(c)]. Vice 22 versa, the 2 seismometers, marked as A0 and A3, were located on the top of the RHS ends (Fig. 4). The 23 seismometer Af was instead placed as a reference to the floor, i.e., near the RHS end at i = 0 [Fig. 3(b)].

1 Af was used to record eventual anomalies of the measuring system. All seismometers were connected to 2 a signal conditioner and, later, to a data logger placed on a desk near the test rig. The experimental set-up 3 in Fig. 4 shows their positions in violet. The thin-walled box-girder's self-mass per unit length (m_{beam}), 4 comprehensive of 4 seismometers (A0, A1, A2 and A3) and horizontal steel plates for positioning a 5 number of displacement transducers, was 0.7167 kN/m. Specifically, such plates, of $300 \times 120 \times 7.5$ mm³ 6 in dimension, 200 mm in length, and made with the same structural steel of the member, were welded 7 along the RHS body [Figs. 5(a)-(b) and Fig. 5(d)]. Notably, the measurements obtained by such 8 transducers were objective of a different study.



Fig. 4. The experimental set-up with the positions of hydraulic jack (*W*) and servo velocity seismometers (Af, A0, A1, A2 and A3). Units: m.



Fig. 5. (a) Hinge support. (b) Roller support. (c) Hydraulic jack (*W*) on a rebar anchored near the midspan and seismometer A2 (Fig. 4). (d) Hydraulic hand pump to the floor before activation.

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11 3.3. Free transverse vibrations

12 Free transverse vibrations were performed in a short term on the thin-walled box-girder following the 13 application of $N_{0x,aver}$, in values of \approx 117, 214, 332, 421, 501 and 598 kN, pulling the cables outward 14 (Section 3.2). The girder was thus subjected to rotational boundary conditions due to the bending moments 15 $N_{0x,aver}$ e at the RHS ends (Fig. 4). All seismometers (Af, A0, A1, A2 and A3) acquired the acceleration 16 data at a sampling rate of 200 Hz, and with a Block Size (BS) of 131,072 samples. Vibration 17 measurements were repeated thrice after that each post-tensioning force $(N_{0x,aver})$ was assigned. 18 18 experiments were totally executed. Particularly, free vibrations were induced pulling out a steel rebar, of 19 6 mm in diameter, welded at a distance not greater than 600 mm from the midspan (i = 2) both at right

1 and left side (Fig. 4). Such procedure was feasible using a hydraulic oil jack W, of 120 mm in diameter, 2 170 mm in length, 100 kN in force capacity, and 118.5 N in mass, placed on the top of the girder and 3 crossed by each rebar [Fig. 5(c)]. These latter were connected to the jack (W) at their higher end. 4 Specifically, the jack pulled out every rebar from the girder counteracting its force on the girder itself. 5 The jack was in turn actuated by a hydraulic hand pump, of 96.53 MPa in pressure capacity, positioned 6 to the floor [Fig. 5(d)]. Therefore, the thin-walled box-girder with initial second-order curvature, due to 7 the bending moments ($N_{0x,aver} e$), total self-mass ($m_{tot,beam} = m_{beam} \times L = 5.0$ kN), and an axial end constraint, 8 due to the stiffness of the cables [21, 22], was vibrated by small unbalanced forces caused by the 9 unavoidable weld's breaking between rebar and member. In turn, weld's breaking was due to its lowest 10 resistance (Fig. 4). Consequently, the value of the release forces (F_d) were of difficult estimations (Fig. 4) 11 [17]. The vibrations were measured with respect to the RHS horizontal (strong) axis. The post-tensioning 12 forces N_{0x1} and N_{0x2} were measured every second for a total of 200 seconds by a data acquisition unit 13 using a different data logger. The average measurements of N_{0x1} and N_{0x2} for one repetition of vibration 14 tests were listed in Table 2. The cables were never in contact with the surrounding RHS during vibrations 15 (Fig. 4) unlike when the thin-walled box-girder was unloaded ($N_{0x,aver}=0$). In this specific case, 16 measurements were discarded. Notably, the eccentric post-tensioning force, when is assumed as 17 externally applied, causes the yielding stress (f_{yk} = 390 MPa) of the most extreme fibres of midspan cross 18 section at a value of $N_{0x,ULS,0.23} \approx 1,007$ kN (Fig. 4). $N_{0x,ULS,0.23}$ corresponds to $\approx 18.2\%$ of the critical 19 buckling load $N_{\text{crE},1}$ or $N_{\text{crE},\text{shear},1} = 5,543$ kN.

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21 3.4. Vibration frequencies

Figs. 6(a)-(b) show the acceleration from time-histories of the thin-walled box-girder, with corresponding Fast Fourier Transform (FFT) results, when the post-tensioning force $N_{0x,aver} = 214$ kN (Fig. 4). BS = 65,536 samples. Particularly, Fig. 6(a) refers to the instrumented section (A1) at the quarter (*i* = 1), whilst Fig. 6(b) refers to the instrumented section (A2) at the midspan (*i* = 2). The raw signals (acceleration-time data) of each test repetition were imported into MATLAB [33] where a signal processing algorithm eliminated significant electrical noise [16]. The peak picking method was then adopted. Fundamental and second-mode frequencies were located at each peak of the FFT, as indicated in Figs. 6(a)–(b). Same procedure was done assuming a BS = 131,072 samples. A total of 72 of which were gathered, since each free vibration was repeated thrice, i.e., when the girder was subjected to each $N_{0x,aver}$. Figs. 6(a)–(b) appertain both to one test repetition. Notably, the maximum $N_{0x,aver,max} = 598$ kN was $\approx 59\%$ of the maximum allowable $N_{0x,ULS,0.23} \approx 1,007$ kN (Section 3.3).



samples. Seismometers: (a) A1; (b) A2 (Fig. 4).

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6 Seismometers A1 and A2 provided a slight increasing in fundamental frequency $f_{1,exp}$ (almost 7 constant) from 18.37 to 18.39 Hz into the initial increment in post-tensioning force ($N_{0x,aver}$) which went 8 from 117 to 214 kN. Subsequently, the girder was subjected to a significant softening effect: the frequency 9 $(f_{1,exp})$ underwent a prominent decrement of 0.93 Hz under an increment in $N_{0x,aver}$ which went up to 598 10 kN. A maximum $f_{1,exp,max} = 18.39$ Hz and a minimum frequency $f_{1,exp,min} = 17.46$ Hz were finally measured 11 within the experimental tendency. All frequencies $(f_{1,exp})$ identified using seismometers A1 and A2, and a 12 BS = 65,536 samples within the FFT of the 3 repetitions, were listed in Tables 2–3. The trend of $f_{1,exp}$ was 13 confirmed assuming a BS = 131,072 samples, i.e., increasing the accuracy in frequency estimations (200 14 Hz / 131,072 = 0.002 Hz). The only difference was the measurement $f_{1,exp}$ = 17.91 Hz when the applied 15 post-tensioning force $N_{0x,aver}$ = 332 kN. Besides, seismometers A1 and A2 always furnished equal 16 measurements $(f_{1,exp})$ in relation to the same configuration of free vibration test (Section 3.3). Notably, the 17 masses composed of load cell, threaded bars, circular and rectangular steel plate at one RHS end [Fig. 18 3(b)] were only $\approx 8.9\%$ of the girder's total self-mass ($m_{tot,beam} = 5.0$ kN). Thus, conversely to what

happened in the experiments of Noble et al. [14], it was avoided that such masses, lower than the total self-mass ($m_{tot,beam}$), affected the beam dynamics. Contrariwise, the second-mode frequencies were affected by the application of weak release forces (F_d) (Fig. 4). To obtain reliable second-mode frequency measurements through FFT, strong excitations are generally required to be induced during vibration testing [17]. Seismometer A1 was strategically located at a distance of 1.745 m from the midspan (at i =1) since the latter is a nodal point for the second-mode vibration of a beam.

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4. Interpretation of experimental results

9 4.1. Comparison with reference solutions

10 The fundamental frequencies ($f_{1,exp}$) were firstly compared with Eqs. (1)–(2) and Eq. (4), in which, n = 1, 11 whereas the corresponding critical buckling loads, $N_{crE,1}$ and $N_{crE,shear,1}$, were respectively assumed 12 (Section 2). The comparisons were presented in Table 2, in which percentage errors were formulated by 13 $\Delta = (f_1 - f_{1,exp}) / f_{1,exp}$. Specifically, $f_{E-B,1,II}$ decreased from ≈ 18.91 to 18.42 Hz into the increment in post– 14 tensioning force ($N_{0x,aver}$) went from ≈ 300 to 598 kN [Eq. (1)] corresponding to an average error 15 Δ_{aver} =5.5%. Vice versa, $f_{\text{shear},1,II}$ decreased from \approx 18.80 to 18.32 Hz into the same range [Eq. (4)] 16 corresponding to an average error $\Delta_{aver}=4.8\%$. The frequencies ($f_{1,exp}$) followed the significant decreasing 17 tendency of Eq. (1) and Eq. (4) and concurrent with the compression-softening theory. Therefore, the 18 trend of frequencies $(f_{1,exp})$ disagreed with findings and deductions of Noble et al. [14]. Yet, the differences 19 between frequencies $f_{\text{E.-B.},1,II}$ and $f_{\text{shear},1,II}$ of ≈ 0.10 Hz (Table 2) were caused by the effect of shear 20 deformation in Eq. (4). By contrast, the slight initial increment in $f_{1,exp}$ of 0.02 Hz, i.e., into the range 117 21 $kN \le N_{0x,aver} \le 214$ kN, was not of the same rate as predicted by Eq. (1) and Eq. (4). Such trend occurred 22 under a lower amount of second-order effects ($\leq 4.0\%$ of $N_{crE,1}$ or $N_{crE,shear,1}$), i.e., without a considerable 23 softening effect along the thin-walled girder. Notably, the total self-mass per unit length (m_{tot}) considered 24 the mass of hydraulic jack, W = 118.5 N [Fig. 5(c)], which was distributed along the length (L) of the 25 member $[m_{tot} = m_{beam} + (W/L) = 0.7337 \text{ kN/m}]$. W was equal to 2.4% of the total self-mass $(m_{tot,beam} = 5.0 \text{ km})$ 26 kN). Based also on published works and previous results [12, 17, 34, 35], masses of sensors or devices 27 (affixed transversally) affect the fundamental frequency $(f_{1,exp})$ of a simply supported beam when are 28 greater than $\approx 0.3\%$ of its total self-mass. This consideration is valid when strong excitations are induced

by an impact or release force (F_d) because the precision in detection of frequency ($f_{1,exp}$) is related to the complexity of signal's response, and to the proportion of vibrational response assigned to the fundamental mode [14, 15, 17]. FE analyses in Strand7 [36], discretizing the thin-walled box-girder into 20 elements, and accounting for total self-mass per unit length (m_{tot}), girder's axial deformations and curvatures, depending on post-tensioning ($N_{0x,aver}$) and bending moments ($N_{0x,aver} e$) at the RHS ends (Fig. 4), furnished the same frequencies $f_{E,-B,,1,II}$. The same values of frequencies $f_{shear,1,II}$ were also obtained including shear deformation in the FE model.

8 9

4.2. Comparison with the FE model proposed by Jaiswal including shear deformation

10 The frequencies $(f_{1,exp})$ were secondly compared with the FE model proposed by Jaiswal [9] in which, the 11 post-tensioning force ($N_{0x,aver}$) is applied in the form of initial tension in the cables [17, 20]. Here $N_{0x,aver}$ 12 becomes an integral part of the thin-walled girder-cable system. As a result, N_{0xaver} is not treated as a 13 compressive force, as takes place in Eq. (1) and Eq. (4). Elastic material, geometric properties and self-14 masses per unit length m_{beam} and m_{tendon} (Table 1 and Section 3.1) were accounted for the corresponding 15 FE model in Strand7 [36]. Transverse shear deformation was considered, whilst longitudinal vibrations, warping and rotary inertia were neglected [37]. Particularly, the thin-walled box-girder was discretized 16 17 into 20 elements, whilst the 5 strands were modeled using a straight tendon (Atendon) in turn discretized 18 into 2 truss elements [Fig. 7(a)]. The tendon was aligned to the girder axis and only had common nodes 19 at the RHS ends. Here, the eccentricity (e) was the distance of the tendon from the girder's center-line, as 20 shown in Fig. 7(a). In such FE model for steel members with an eccentric unbonded tendon, only end 21 nodes of beam and tendon are connected by rigid links [Fig. 7(a)]. The mass of the hydraulic jack, W =22 118.5 N [Fig. 5(c)], was first accounted as a distributed load $[m_{tot} = m_{beam} + (W/L)]$. Second, the jack was 23 accounted as a concentrated mass (W) modeled as a short beam element and connected at the midspan 24 [Fig. 7(a)]. To obtain the FE frequencies (f_{E.-B.,1,II,FE}, f_{shear,1,II,FE}, f_{E.-B.,1,II,FE,W}, f_{shear,1,II,FE,W}), a two-step 25 approach was performed. The post-tensioning force $(N_{0x,aver})$ was applied in the tendon, while axial 26 deformations and second-order curvatures of the systems were gained. Artificial geometric imperfections 27 were not included to prevent favoring instability modes. Then, the FE frequencies were evaluated for the 28 corresponding new deformed configurations. Notably, the cross sectional area of strands (A_{tendon}) was

assumed as unchanged after deformations. Vice versa, the deformed configurations related to the firstorder FE frequencies ($f_{E-B,1,I,FE}$, $f_{shear,1,I,FE}$, $f_{E-B,1,I,FE,W}$, $f_{shear,1,I,FE,W}$) were only caused by the self-masses of girder, tendon and jack (*W*). Yet, an additional FE model was generated in which the tendon (A_{tendon}) was connected to the surrounding RHS by one rigid link at the midspan, as depicted in Fig. 7(b), thus simulating the presence of a deviator. Figs. 7(a)–(b) show the two fundamental flexural shapes $v^{(0)}$ of the thin-walled box-girder with concentrated mass (*W*) at the midspan: it is clearly visible that only the RHS ends are governed by the conservation of the planarity of the cross sections.





8

9 The comparisons are listed in Table 3; percentage errors $\Delta = (f_{1,\text{FE}} - f_{1,\text{exp}}) / f_{1,\text{exp}}$. When the total 10 self-mass per unit length (m_{tot}) was accounted, the FE frequencies $f_{\text{E-B},1,II,\text{FE}}$ decreased from ≈ 18.95 to 11 18.50 Hz into the increment $\approx 300 \le N_{0x,aver} \le 598$ kN, corresponding to an average error $\Delta_{aver} = 5.8\%$, 12 whereas $f_{\text{shear,1,II,FE}}$ decreased from ≈ 18.84 to 18.38 Hz into the same range, corresponding to an error Δ_{aver} 13 = 5.2%. Conversely, when the jack was accounted as a mass (W), the FE frequencies $f_{\text{E-B},1,II,\text{FE},W}$ decreased 14 from ≈ 18.73 to 18.29 Hz into $\approx 300 \le N_{0x,aver} \le 598$ kN, corresponding to an error $\Delta_{aver} = 4.6\%$, whereas 15 $f_{\text{shear},1,II,\text{FE},W}$ decreased from ≈ 18.62 to 18.17 Hz, corresponding to an error $\Delta_{\text{aver}} = 4.0\%$. The experimental 16 frequencies $(f_{1,exp})$ followed the decreasing tendency of the aforementioned FE frequencies (Table 3). 17 Consequently, they disagreed with findings and deductions of Noble et al. [14]. Similarly to what 18 predicted by the reference solutions (Section 4.1), the slight initial increment in frequency ($f_{1,exp}$) of 0.02 19 Hz was not of the same rate. Generally, the FE model, which included shear deformation, and accounted 20 for the jack as a mass (W) at the midspan ($f_{\text{shear},1,II,\text{FE},W}$), better simulated the compression-softening effects. 21 Table 3 also displays the trend of FE frequency ($f_{\text{shear},1,II,\text{FE},W,D}$) with the presence of a deviator [Fig. 7(b)]. 22 In this configuration, the compression-softening effect is eliminated when the tendon is in contact with the surrounding thin-walled section: $f_{\text{shear},1,II,\text{FE},W,D}$ slightly decreased from 18.52 to 18.34 Hz into the increment $0 \le N_{0x,\text{aver}} \le 598$ kN. Notably, the FE model considered the slight influence of the bending moments at the pinned-end supports (Fig. 4), $M_{\mu} = \mu_{\text{sliding}} R \times h/2 = \mu_{\text{sliding}} \{ [(m_{\text{tot}} + m_{\text{tendon}}) \times L]/2 \} \times h/$ 2, caused by the dynamic sliding friction between steel-on-steel. A frictional coefficient (μ_{sliding}) of 0.42, regarding the contact between dry steel-on-steel surfaces, was assumed [38].

6 The correlations summarized in Tables 2–3 confirmed the use of an accurate value of fundamental 7 frequency $(f_{1,exp})$ as indicator for axial load estimation in steel beams, bridge cables and hangers [34, 39– 8 42]. Yet, this work made certain such use for post-tensioning loss identification in thin-walled steel-9 girder-bridges with straight cables that are not in contact with the surrounding section [1, 43, 44]. In fact, 10 second-order effects are not negligible when transverse vibrations are induced: a decrement in post-11 tensioning force ($N_{0x,aver}$) of 100 kN corresponds to an increment in $f_{1,exp}$ of ≈ 0.18 Hz. By contrast, when 12 the cables are in contact with the section or draped with a deviator [43, 45, 46], its use as a parameter was made doubtful: a decrement in $N_{0x,aver}$ of 100 kN corresponds to an increment in $f_{1,exp}$ of 0.03 Hz. 13 14 Regarding PC girder-bridges, the frequency $(f_{1,exp})$ was instead deduced to be a suitable indicator for 15 flexural rigidity determination because the dynamics of a pre or post-tensioning force is experimentally 16 negligible with a straight tendon [11, 47, 48] whilst is almost neglectable with a parabolic one [8, 12, 16].

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18 4.3. Comparison with FE plate and shell modeling

FE plate and shell modeling was thirdly conducted to numerically compare the frequencies ($f_{1,exp}$). The 19 20 spatial solution of the thin-walled steel-box-girder was discretized into a group of FE linked to each other 21 in a certain manner. Since basic elements can be combined in different ways, FE analyses can solve 22 problems of dynamics of thin-walled structures involving complicated geometries and/or boundary 23 conditions [49, 50]. Elastic material, geometric properties, self-masses and unit weights (Table 1 and 24 Section 3.1) were accounted in Strand7 [36]. The thin-walled box-girder was modeled using 276,860 plate/shell elements with a mesh size= 5×5 mm². The 5 straight strands were discretized into 2 truss 25 26 elements per each and fixed at the RHS ends according to the member design [Fig. 3(b)]. The plates for transducers (24,000 elements), internal stiffeners (6,948 elements) and plates at the RHS ends (3,846 27 28 elements) were also modeled using plate/shell elements with the same mesh size. The upper part of

1 supports of 100 mm in width, 60 mm in height and 141 mm in length (13,536 elements), load cells (24,422 2 elements) and circular plates (12,211 elements) were instead modeled using brick elements with a mesh 3 size= $5 \times 5 \times 5$ mm³. The meshes between thin-walled girder, strands, plates and load cells were rigidly 4 connected at the RHS ends. The seismometers A0, A1, A2 and A3 and the jack (W) [Fig. 5(c)] were 5 modeled as short beams and connected along the girder according to the test layout (Fig. 4). Young's 6 modulus of supports, load cells, circular plates, seismometers and jack (W) was assumed of 200 GPa, 7 whilst v = 0.25. Carriage-hinge external constraints, to have the pinned-end conditions, were arranged 8 on a row of nodes at L (Fig. 4) whilst longitudinal vibrations, warping and rotary inertia were neglected. 9 To obtain the FE frequencies ($f_{1,II,FE,P\&S}$), the two-step procedure was conducted (Section 4.2). The post– tensioning force $(N_{0x,aver})$ was applied in the strands and the axial deformation of the girder-tendon system 10 11 was gained. Artificial geometric imperfections were not included. Subsequently, $f_{1,II,FE,P\&S}$ were achieved. 12 Yet, an additional FE model was created to simulate the contact between strands and surrounding RHS using one deviator, of 6 mm in thickness, at the midspan. 1,923 plate/shell elements with a mesh size=5×5 13 14 mm² were adopted ($\rho_{beam} = 77.01 \text{ kN/m}^3$; E = 205.12 GPa) [Fig. 8(b)]. In this case, the FE frequencies 15 were labeled as $f_{1,II,FE,P\&S,D}$. Fig. 8(a) and Fig. 8(c) respectively show the FE fundamental flexural shape 16 of the thin-walled box-girder without and with deviator for $N_{0x,aver,max} = 598$ kN.



flexural shape for $N_{0x,aver,max}$ =598 kN; (b) Deviator at the midspan; (c) Fundamental flexural shape with deviator ($N_{0x,aver,max}$ =598 kN).

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18 Comparisons were reported in Table 2 [$\Delta = (f_{1,FE,P\&S} - f_{1,exp}) / f_{1,exp}$]. $f_{1,II,FE,P\&S}$ decreased from \approx 18.61 to 19 18.14 Hz corresponding to an average error $\Delta_{aver} = 3.8\%$. Therefore, the dynamics of the thin-walled box-20 girder was numerically well described since the experimental frequencies ($f_{1,exp}$) properly followed the 21 decreasing tendency of the compression-softening theory into the range $\approx 300 \le N_{0x,aver} \le 598$ kN. 1 Conversely, $f_{1,II,FE,P\&S,D}$ slightly decreased from 18.51 to 18.33 Hz into the increment $0 \le N_{0x,aver} \le 598$ kN 2 (Table 2). It was confirmed that the compression-softening effect is canceled when the cables are in 3 contact with the surrounding thin-walled section (Section 4.2). Likewise, the FE plate and shell modeling 4 always considered the dynamic sliding friction ($\mu_{sliding} R$) at the girder's pinned-end supports (Fig. 4) 5 because of the contact between dry steel-on-steel surfaces.

Table 2. Comparison between measured $(f_{1,exp})$ and numerical fundamental frequencies from

reference solutions [Eqs. (1), (2) and (4)] and FE plate and shell modeling. Seismometers A1 and A2 for BS

= 65,536 samples (Fig. 4).

6 7

8

				Reference Solutions			FE Plate and Shell Modeling			
				Euler-Bernoulli	She	Shear		Shear		a Deviator
				II–ord.	I-ord.	II-ord.	I-ord.	II-ord.	I-ord.	II-ord.
No.1	No.2	No	fi	fЕВ.,1, <i>II</i>	$f_{\mathrm{shear},1,I}$	$f_{\mathrm{shear},1,II}$	filter	fi ure nec	fillener	fi ure nic n
TV0x1	1V0x1 1V0x2	1 V0x, aver	J1,exp	Eq. (1)	Eq. (2)	Eq. (4)	J 1,1,FE,P&S	J1,11,FE,P&S	J1,1,FE,P&S,D	J1,11,FE,P&S,D
(kN)	(kN)	(kN)	(Hz)	(Hz)	(Hz)	(Hz)	(Hz)	(Hz)	(Hz)	(Hz)
		0		19.50	19.39	-	19.18	-	18.51	_
_	_	0	—	-	-	_	-	-	-	-
111	122	117	18.37	19.30	19.39	19.19	19.18	18.98	18.51	18.47
111	122	117	-	(5.1%)	(5.6%)	(4.5%)	(4.4%)	(3.3%)	-	_
205	222	214	18.39	19.12	19.39	19.01	19.18	18.82	18.51	18.45
203	LLL	214	_	(4.0%)	(5.4%)	(3.4%)	(4.3%)	(2.3%)	-	-
272	241	222	17.94	18.91	19.39	18.80	19.18	18.61	18.51	18.41
525	341	332	-	(5.4%)	(8.1%)	(4.8%)	(6.9%)	(3.7%)	-	_
410	422	421	17.78	18.75	19.39	18.64	19.18	18.45	18.51	18.38
410	432	421	-	(5.5%)	(9.1%)	(4.8%)	(7.9%)	(3.8%)	-	_
400	510	501	17.64	18.60	19.39	18.49	19.18	18.31	18.51	18.36
490	312	501	_	(5.4%)	(9.9%)	(4.8%)	(8.7%)	(3.8%)	-	_
506	610	500	17.46	18.42	19.39	18.32	19.18	18.14	18.51	18.33
380	010	398	_	(5.5%)	(11.1%)	(4.9%)	(9.9%)	(3.9%)	-	-
				$m_{\rm tot} = m_{\rm beam} + (W/L)$			mbean	h_n and $W \to M$	Mass W conce	entrated

9 10

11

Table 3. Comparison between measured ($f_{1,exp}$) and numerical fundamental frequencies from the FE model [9]. Seismometers A1 and A2 for BS = 65,536 samples (Fig. 4).

		FE Model proposed by Jaiswal									
		Euler-Bernoulli		Shear		Euler-H	Bernoulli	Shear		Shear with	a Deviator
		I-ord.	II-ord.	I-ord.	II-ord.	I-ord.	II-ord.	I–ord.	II-ord.	I–ord.	II–ord.
N _{0x,aver}	$f_{1,\exp}$	<i>f</i> _{EB.,1,<i>I</i>,FE}	f _{EB.,1,<i>II</i>,FE}	$f_{\rm shear, 1, I, FE}$	fshear,1,11,FE	f _{EB.,1,<i>I</i>,FE,W}	f _{EB.,1,<i>II</i>,FE,<i>W</i>}	$f_{\mathrm{shear},1,I,\mathrm{FE},W}$	$f_{\mathrm{shear},1,II,\mathrm{FE},W}$	f _{shear,1,<i>I</i>,FE,W,D}	fshear,1,11,FE,W,D
(kN)	(Hz)	(Hz)	(Hz)	(Hz)	(Hz)	(Hz)	(Hz)	(Hz)	(Hz)	(Hz)	(Hz)
0		19.50	-	19.39	-	19.28	-	19.17	_	18.52	-
0	-	-	-	-	-	-	_	-	_	-	_
117	18.37	19.50	19.31	19.39	19.20	19.28	19.09	19.17	18.98	18.52	18.49
11/	-	(6.2%)	(5.1%)	(5.6%)	(4.5%)	(5.0%)	(3.9%)	(4.4%)	(3.3%)	-	-
214	18.39	19.50	19.15	19.39	19.04	19.28	18.93	19.17	18.82	18.52	18.46
214	-	(6.0%)	(4.1%)	(5.4%)	(3.5%)	(4.8%)	(2.9%)	(4.2%)	(2.3%)	-	-
222	17.94	19.50	18.95	19.39	18.84	19.28	18.73	19.17	18.62	18.52	18.42
332	-	(8.7%)	(5.6%)	(8.1%)	(5.0%)	(7.5%)	(4.4%)	(6.9%)	(3.8%)	-	-
421	17.78	19.50	18.80	19.39	18.69	19.28	18.59	19.17	18.47	18.52	18.40
421	-	(9.7%)	(5.7%)	(9.1%)	(5.1%)	(8.4%)	(4.6%)	(7.8%)	(3.9%)	-	-
501	17.64	19.50	18.66	19.39	18.55	19.28	18.45	19.17	18.34	18.52	18.37
301	-	(10.5%)	(5.8%)	(9.9%)	(5.2%)	(9.3%)	(4.6%)	(8.7%)	(4.0%)	-	-
508	17.46	19.50	18.50	19.39	18.38	19.28	18.29	19.17	18.17	18.52	18.34
598	-	(11.7%)	(6.0%)	(11.1%)	(5.3%)	(10.4%)	(4.8%)	(9.8%)	(4.1%)	-	_
			$m_{\rm tot} = m_{\rm bea}$	m + (W/L)		m_{beam} and $W \rightarrow \text{Mass } W$ concentrated					

- **5. Experiments performed by Noble et al.**
- 5.1. Free transverse vibration of two thin-walled steel-box members post-tensioned by a concentric
 straight cable

4 In Noble et al. [14], the assumption that a post-tensioning force is whether or not dynamically equivalent 5 to a compressive force was investigated (Section 1). To make sure that findings and deductions on our 6 tests were effective (Section 4), numerical validations of the experiments performed by Noble et al. [14] 7 were additionally executed. Specifically, the researchers carried out free transverse vibrations on two thin-8 walled steel members with RHS post-tensioned by a concentric straight cable. Beam 1 had b = 50 mm in 9 width, h = 30 mm in height, and $t = t_I = 3$ mm in thickness, whereas Beam 2 had b = 120 mm, h = 60 mm, 10 and $t = t_1 = 3$ mm. Their total self-masses per unit length ($m_{tot} = \rho_{beam} \times A$) were 0.0343 and 0.0806 kN/m, 11 respectively. The steel-box members were post-tensioned by a concentric steel "seven-wire" strand of 12 15.7 mm in diameter, whereas their material and geometric properties were summarized in Table 4. The 13 self-mass per unit length of the strand ($m_{\text{tendon}} = \rho_{\text{tendon}} \times A_{\text{tendon}}$) was instead 0.0114 kN/m. It was anchored 14 with collets either side of two hydraulic jacks of 30 tons. in force capacity, and 222.7 N in mass per each 15 (RCD307, Simplex). A load cell of 85 mm in diameter, 80 mm in length, 300 kN in force capacity, 1.5 16 mV/V in sensitivity, and 19.6 N in mass, was instead fixed at the extremities of the two jacks to measure 17 the applied post-tensioning forces ($N_{0x,aver}$) (KCM-300KNA, Tokyo Measuring Instruments Lab.). 18 Moreover, a 200×150×20 mm³ steel plate, of 77.01 kN/m³ in unit weight, 200 GPa in Young's modulus, 19 and 80 GPa in shear one, was positioned between each jack and the RHS end. Also, the members were 20 pinned-end by steel knife-edge supports in a test rig, resulting in a clear length L (Table 4). One of the 21 jacks was connected to a hydraulic hand pump to apply a post-tensioning force $(N_{0x,aver})$ by pulling the 22 strand outward. Thus, impact hammer testing were executed at incremental levels of $N_{0x,aver}$ [Fig. 3 in 23 Noble et al. [14]]. In Beam 1, the maximum induced second-order effects were $\approx 89\%$ of its critical 24 buckling load $N_{crE,1}$ or $N_{crE,shear,1}$ = 56 kN. Differently, in Beam 2, maximum second-order effects were \approx 25 30% of its $N_{crE,1}$ or $N_{crE,shear,1}$ = 597 kN. The strand was only in contact at the edges, but not throughout the RHS length during vibrations. Therefore, the eccentricity (e) of $N_{0x,aver}$ was theoretically zero (Fig. 1). 26 27 Following execution of hammer testing, acceleration-time data, with respect to the specimens' horizontal axis, were acquired by an accelerometer of 6 g in mass, and 0.05 mV(m/s²)⁻¹ in sensitivity, which was 28

1	vertically arranged at the midspans (3200B4, Dytran Instruments) [Figs. 7 and 9 in Noble et al. [14]]. The
2	FFT was then performed on the acceleration-time data to describe the signal in the frequency domain.
3	And, peak picking method was used to identify the fundamental frequencies $(f_{1,exp})$ (Section 3.4). The
4	frequencies $(f_{1,exp})$ were determined for each increment of post-tensioning force $(N_{0x,aver})$ and each hammer
5	test conducted [Figs. A1(b) and A1(d) in Noble et al. [14]]. More information on experiments and modal
6	analyses are reported in Noble et al. [14]. The applied post-tensioning forces ($N_{0x,aver}$) and average
7	identified frequencies $(f_{1,exp})$ were also listed in Table 5. For both specimens, a slight decreasing trend in
8	frequency $(f_{1,exp})$ was observed, which was not of the same rate as predicted by the Euler-Bernoulli theory
9	[Eq. (1)]. In Beam 1, $f_{1,exp}$ decreased from 27.77 to 24.77 Hz into the increment in post-tensioning force
10	$(N_{0x,aver})$ which went from 0 to 50 kN. Vice versa, in Beam 2, $f_{1,exp}$ decreased from 75.35 to 72.85 Hz into
11	an increment which went up to 180 kN. Regression lines were fitted to the data ($f_{1,exp}$) and depicted in
12	blue in Figs. 9(a)-(b). Accordingly, Noble et al. [14] concluded that a post-tensioning force is not
13	dynamically equivalent to a compressive force. Strictly, the researchers deduced that the compression-
14	softening theory is not applicable to prestressed structures. Based on our findings and deductions (Section
15	4), it is questionable that the frequencies $(f_{1,exp})$ of their post-tensioned box members did not significantly
16	follow the decreasing trend of the compression-softening theory [Figs. A1(b) and A1(d) in Noble et al.
17	[14]].

 Table 4. Noble et al. [14]. Properties of Beam 1 and Beam 2.

Property	Beam 1) RHS $50 \times 30 \times 3 \text{ mm}^3$	Beam 2) RHS $120 \times 60 \times 3 \text{ mm}^3$
$I (\mathrm{mm}^4)$	6.1812×10^4	6.64092×10^5
$A \text{ (mm}^2)$	444	1,044
$A_w (\mathrm{mm}^2)$	144	324
Slenderness	127	59
<i>k</i> [Eq. (3)]	0.321	0.251
<i>L</i> (m)	1	1.5
$ ho_{\text{beam}}$ (kN/m ³)	77	7.20
E (GPa)	2	205
f_{yk} (MPa)	2	235
G (GPa)	:	82
υ	0	.25
Atendon (mm ²)	1	.49
ρ_{tendon} (kN/m ³)	76	5.62
Etendon (GPa)	2	200
σ_{uy} (MPa)	1,	860

1 5.2. Comparison with reference solutions

2 Following the numerical approaches described in Section 4, Noble et al. [14] 's fundamental frequencies 3 $(f_{1,exp})$ were firstly compared with Eq. (1) and Eq. (4) in which n = 1, whereas the critical buckling loads 4 of Beams 1 and 2, N_{crE,1} and N_{crE,shear,1}, were assumed (Section 2). Comparisons were presented in Figs. 5 9(a)–(b): the trends of frequencies $f_{E-B,1,II}$ were depicted in pink [Eq. (1)], whereas those of frequencies 6 $f_{\text{shear,1,II}}$ were depicted in red with cross symbols [Eq. (4)]. In Beam 1, $f_{\text{E-B-1,II}}$ decreased from 42.04 to 7 13.32 Hz into the increment in post-tensioning force ($0 \le N_{0x,aver} \le 50$ kN), whereas $f_{shear,1,II}$ decreased 8 from 41.94 to 13.29 Hz [Fig. 9(a)]. In Beam 2, $f_{E-B,1,II}$ decreased from 89.87 to 75.11 Hz into the increment 9 $0 \le N_{0x,aver} \le 180$ kN, whereas $f_{shear,1,II}$ decreased from 88.64 to 74.09 Hz [Fig. 9(b)]. Therefore, the frequencies $f_{1,exp}$, underlined by the blue regression lines, did not follow the significant decreasing 10 11 tendency of the compression-softening theory of Eq. (1) and Eq. (4). The experimental behavior of the 12 post-tensioned steel-box members was not well described confirming findings and deductions of Noble 13 et al. [14]. The considerable softening effect, in terms of second-order effects, is particularly visible in Beam 1 ($\approx 45\%$ of its $N_{\text{crE},1}$ or $N_{\text{crE},\text{shear},1}$). Indeed, the frequencies $f_{\text{E}-\text{B},1,II}$ and $f_{\text{shear},1,II}$ started to converge 14 15 to the buckling loads ($N_{crE,1}$ and $N_{crE,shear,1}$) at a level of post-tensioning force ($N_{0x,aver}$) of ≈ 25 kN. Notably 16 also that, in Fig. 9(a), shear deformation is visibly negligible due to the high slenderness of Beam 1 equal 17 to 127.





frequencies from FE plate and shell modeling.									
	Be	eam 1) RHS	$50 \times 30 \times 3$ mm	1 ³	Beam 2) RHS $120 \times 60 \times 3 \text{ mm}^3$				
		Shear	Shear with	Shear with			Shear	Shear with	Shear with
			1 Rigid Link	3 Rigid Links				1 Rigid Link	3 Rigid Links
		II–ord.	II–ord.	II–ord.			II–ord.	II–ord.	II–ord.
N _{0x,aver}	$f_{1,\exp}$	<i>f</i> 1, <i>11</i> ,FE,P&S	$f_{1,II,FE,P\&S,1Link}$	$f_{1,II,\text{FE},\text{P\&S},3\text{Links}}$	N _{0x,aver}	$f_{1,\exp}$	<i>f</i> 1, <i>11</i> ,FE,P&S	<i>f</i> 1, <i>II</i> ,FE,P&S,1Link	$f_{1,II,\text{FE},\text{P\&S},3\text{Links}}$
(kN)	(Hz)	(Hz)	(Hz)	(Hz)	(kN)	(Hz)	(Hz)	(Hz)	(Hz)
0	32.50	30.49	28.42	33.37	0	75.00	74.48	70.78	75.00
0	_	(-6.2%)	(-12.6%)	(2.7%)	0	_	(-0.7%)	(-5.6%)	(0.0%)
5	24.50	29.76	28.27	33.35	20	73.75	73.52	70.59	73.87
5	_	(21.5%)	(15.4%)	(36.1%)	20	_	(-0.3%)	(-4.3%)	(0.2%)
10	24.00	29.02	28.13	33.32	40	75.50	72.52	70.39	74.29
	_	(20.9%)	(17.2%)	(38.8%)	40	_	(-3.9%)	(-6.8%)	(-1.6%)
15	25.50	28.26	27.98	33.30	60	78.75	71.56	70.22	74.33
	_	(10.8%)	(9.7%)	(30.6%)		_	(-9.1%)	(-10.8%)	(-5.6%)
20	27.50	27.39	27.83	33.28	80	72.75	70.52	70.00	74.30
20	_	(-0.4%)	(1.2%)	(21.0%)		_	(-3.1%)	(-3.8%)	(2.1%)
25	28.00	26.59	27.68	33.26	100	71.25	69.39	70.36	74.53
23	_	(-5.0%)	(-1.1%)	(18.8%)		_	(-2.6%)	(-1.2%)	(4.6%)
20	26.00	25.74	27.53	33.23	120	74.25	68.42	69.64	74.30
50	_	(-1.0%)	(5.9%)	(27.8%)	120	_	(-7.9%)	(-6.2%)	(0.1%)
25	25.50	24.84	27.37	33.21	140	72.75	67.35	69.43	74.22
33	_	(-2.6%)	(7.3%)	(30.2%)	140	_	(-7.4%)	(-4.6%)	(2.0%)
40	25.00	23.90	27.22	33.19	160	74.25	66.27	69.22	74.17
40	_	(-4.4%)	(8.9%)	(32.8%)	100	_	(-10.7%)	(-6.8%)	(-0.1%)
15	25.75	22.92	27.06	33.16	180	72.75	65.16	69.02	74.10
43	_	(-11.0%)	(5.1%)	(28.8%)	180	_	(-10.4%)	(-5.1%)	(1.9%)
50	24.75	21.89	26.90	33.14					
50	_	(-11.6%)	(8.7%)	(33.9%)					

Table 5. Noble et al. [14]. Comparison between experimental $(f_{1,exp})$ and numerical fundamental
frequencies from FE plate and shell modeling.

1 2

4 5.3. Comparison with the FE model proposed by Jaiswal including shear deformation

5 Noble et al. [14] 's fundamental frequencies ($f_{1,exp}$) were secondly compared with the FE model proposed 6 by Jaiswal [9] which was created following the approaches described in Section 4.2. Elastic material, 7 geometric properties and self-masses per unit length (m_{tot} and m_{tendon}) of the two post-tensioned box 8 members (Table 4 and Section 5.1) were accounted for the corresponding FE models in Strand7 [36]. 9 Particularly, the straight cable was aligned to the center-line of the members, and only connected to the 10 end nodes of the RHS as shown in Fig. 10(a). Besides, Beams 1 and 2 were discretized into 20 elements, whilst cables were discretized into 2 truss elements [Fig. 10(a)]. To obtain the FE frequencies (fshear, 1, II, FE), 11 12 the two-step approach was performed. The post-tensioning force $(N_{0x,aver})$ was applied in the cable and 13 the axial static deformation of each system was achieved. Then, the FE frequencies ($f_{\text{shear},1,II,\text{FE}}$) were 14 gained for the corresponding new deformed configurations. Notably, the cross sectional area of cables 15 (Atendon) was assumed as unchanged after deformations. Fig. 10(b) shows the FE fundamental flexural

- 1 shape of Beam 2 for $N_{0x,aver,max}$ =180 kN [Fig. 9(b)]: it is clearly visible that the cable remains horizontal,
- 2 thus only the ends of the member-cable system are governed by the conservation of the planarity of the
- 3 cross sections.





5 The comparisons were presented in Figs. 9(a)–(b): the trends of FE frequencies $f_{\text{shear},1,II,\text{FE}}$ were 6 depicted in brown. In Beam 1, $f_{\text{shear,1,II,FE}}$ decreased from 41.96 to 25.39 Hz into the increment $0 \le N_{0x,\text{aver}}$ 7 \leq 50 kN [Fig. 9(a)]. Conversely, in Beam 2, $f_{\text{shear},1,II,\text{FE}}$ decreased from 88.96 to 76.47 Hz into $0 \leq N_{0x,\text{aver}} \leq$ 8 180 kN [Fig. 9(b)]. We see again that the frequencies $f_{1,exp}$ did not follow the decreasing tendency of 9 f_{shear,1,II,FE}. Accordingly, the dynamics of the two post-tensioned members was not properly described. 10 The softening effects were lower than those simulated by Eq. (1) and Eq. (4) because the cables increased 11 the stiffness of each member-cable system. This behavior was extremely evident in Beam 1 having a high 12 slenderness [Fig. 9(a)]. Notably, the FE models always considered the sliding friction ($\mu_{\text{sliding}}R$) at the 13 members' supports because of the contact between dry steel-on-steel surfaces (Section 4.2).

14

4

15 5.4. Comparison with FE plate and shell modeling

16 FE plate and shell modeling was thirdly conducted to numerically compare Noble et al. [14] 's frequencies 17 $(f_{1,exp})$. The FE approaches illustrated in Section 4.3 were adopted in Strand7 [36]. Each cable was 18 discretized into 2 truss elements. Beams 1 and 2 were respectively modeled using 7,680 and 17,280 plate/shell elements corresponding to a mesh size of 5×6.25 mm². Instead, the two steel plates were 19 20 modeled using 9,600 brick elements for a mesh size= $5 \times 5 \times 5$ mm³. The meshes of thin-walled box 21 members, cables and plates were rigidly connected at the RHS ends. Unit weights and elastic material 22 properties were described in Table 4 and Section 5.1. Carriage-hinge external constraints were arranged 23 on a row of nodes at L (Table 4). To obtain the FE frequencies ($f_{1,II,FE,P\&S,NoMasses}$), the two-step procedure 1 was similarly conducted. The post-tensioning force ($N_{0x,aver}$) was applied in the cable and the axial 2 deformation of each system was gained. Subsequently, the FE frequencies ($f_{1,II,FE,P\&S,NoMasses}$) were 3 achieved. In Beam 1, $f_{1,II,FE,P\&S,NoMasses}$ decreased from 46.82 to 34.07 Hz into the increment $0 \le N_{0x,aver} \le$ 4 50 kN [in orange in Fig. 9(a)]. Vice versa, in Beam 2, $f_{1,II,FE,P\&S,NoMasses}$ decreased from 92.20 to 81.11 Hz 5 into $0 \le N_{0x,aver} \le 180$ kN [in orange in Fig. 9(b)]. The softening effects were affected by the steel plates at 6 the RHS ends which increased the stiffness of each member–cable system.

7 The influence of load cells and hydraulic jacks was afterwards investigated: Bonopera et al. [11, 8 12] preliminarily declared that Noble et al. [14] 's frequencies ($f_{1,exp}$) were affected by such masses 9 (Section 1). The load cells (KCM-300KNA), of 60 mm in width, 75 mm in height, 80 mm in length, 10 modeled using 5,760 brick elements, for a mesh size= $5 \times 5 \times 5$ mm³, were accounted with their proper mass 11 (Section 5.1). Conversely, the hydraulic jacks (RCD307), of 100 mm in width, 90 mm in height, 220 mm 12 in length, modeled using 31,680 brick elements, and with the same mesh size, were accounted with a 13 lower mass. In fact, it was found that the jacks were arranged touching the floor at their mid length [Fig. 14 6.16. in Noble [51]]. Accordingly, their mass was hyphotetically reduced of 50% to decrease their 15 dynamic contribution. Young's modulus of load cells and jacks was assumed of 200 GPa, whilst v = 0.25. 16 The meshes between cables, plates, load cells and jacks were rigidly connected. The FE frequencies 17 $(f_{1,II,FE,P\&S})$ were obtained using the two-step procedure. In Beam 1, $f_{1,II,FE,P\&S}$ decreased from 30.49 to 18 21.89 Hz into $0 \le N_{0x,aver} \le 50$ kN [in black with triangles in Fig. 9(a)]. Vice versa, in Beam 2, $f_{1,II,FE,P\&S}$ 19 decreased from 74.48 to 65.16 Hz into $0 \le N_{0x,aver} \le 180$ kN [in black with triangles in Fig. 9(b)]. 20 Subsequently, the contact between straight cable and surrounding RHS of the two box members was 21 additionally simulated. First, it was simplified by one rigid link, which was used to connect cable and 22 RHS upper part at their midspans. Second, it was simplified by three rigid links equidistant fastened along 23 their length (L). In the first condition, for Beam 1, average percentage error $\Delta_{aver,1Link} = (f_{1,II,FE,P\&S,1Link} - f_{1,II,FE,P\&S,1Link})$ 24 $f_{1,exp}$ / $f_{1,exp} = 6.0\%$ [Fig. 9(a)], whilst, for Beam 2, $\Delta_{aver,1Link} = -5.5\%$ [Fig. 9(b)]. In the second condition, 25 for Beam 1, average error $\Delta_{\text{aver,3Links}} = (f_{1,II,\text{FE,P&S,3Links}} - f_{1,\text{exp}}) / f_{1,\text{exp}} = 27.4\%$ [Fig. 9(a)], whilst, for Beam 26 2, $\Delta_{aver,3Links} = 0.4\%$ [Fig. 9(b)]. The errors (Δ) between experiments and FE plate and shell modeling for each $N_{0x,aver}$ were listed in Table 5. Figs. 11(a)–(f) show the FE fundamental flexural shapes of Beams 1 27

1 and 2 according to their maximum post-tensioning ($N_{0x,aver,max}$). Based on the above correlations, it was 2 confirmed that the mass at one end constraint, when is greater than the self-mass of the girder themselves, 3 significantly affects the vibrational dynamics. Moreover, in Noble et al. [14], the cables probably touched 4 the surrounding RHS causing stiffening effects which, in turn, falsified the decreasing tendencies of the 5 compression-softening theory [Figs. A1(b) and A1(d)]. Similarly to what gathered by the FE analyses of 6 our experiments (Sections 4.2 and 4.3), we deduced that the dynamics of a compressive force and that of 7 a post-tensioning is different only when the cables are in contact with the surrounding beam's section. 8 Likewise, the FE modeling always considered the sliding friction ($\mu_{\text{sliding}}R$) at the members' supports 9 because of the contact between dry steel-on-steel surfaces (Section 4.2).



10 11

6. Relationship between free transverse vibration and critical buckling

The effect of the location of straight cables on the relationship between free transverse vibrations and critical buckling including shear deformation was discussed. In order to observe the evolution of fundamental frequency in more detail, Fig. 12 and Fig. 13 compare the stability of thin-walled member– cable systems [Fig. 7(a)] with that of the same girders when externally axially loaded (Fig. 1). In the second case, the stability is determined by the critical force ($N_{0x,aver}$) for which the fundamental frequency drops to zero [23]. For our thin-walled steel-box-girder (Fig. 2), the stability was numerically studied since it was not possible to over post–tensioning the beam in the experimental set-up (Fig. 4). Fig. 12

1 demonstrates that increasing the post-tensioning force ($N_{0x,aver}$), the FE frequency $f_{shear,1,II,FE,W}$ 2 considerably reduces up to $N_{0x,aver}$ of $\approx 1,000$ kN, corresponding to the eccentric post-tensioning force 3 $(N_{0x,ULS,0.23})$ which causes the yielding stress of the most extreme fibres of midspan cross section. 4 Subsequently, $f_{\text{shear},1,II,\text{FE},W}$ increases because of the stiffening effect due to the cables' eccentricity (e/h =5 0.23), i.e., when the thin-walled girder is subjected to higher bending moments at the RHS ends ($N_{0x,aver}$) 6 e) and second-order curvatures [21, 22]. Nonetheless, the eccentric post-tensioning force $(N_{0x,pl,0,23})$, 7 which causes the plasticity of the cross section, is equal to 1,484 kN. Conversely, when the eccentricity 8 of cables is large (e/h = 0.74), the stiffening effect generally prevails and the vibration tendon mode is 9 dominant. In this case, $N_{0x,ULS,0.74}$ and $N_{0x,pl,0.74}$ become respectively equal to 502 and 758 kN (Fig. 12). 10 Yet, when the corresponding initial second-order curvatures of the girder are not imposed as deformed 11 configurations (Section 4.2), f_{shear,1,II,FE,W} follows the decreasing trend of the compression-softening theory 12 which, in turn, does not depend on cables' eccentricity (e), as occurred using FE plate and shell modeling 13 $(f_{1,II,FE,P\&S})$, Eq. (1) or Eq. (4). Specifically, the course of $f_{\text{shear},1,II,FE,W}$ converges to a critical buckling load 14 of 5,890 kN, which is higher than $N_{crE,1}$ or $N_{crE,shear,1} = 5,543$ kN (Fig. 12), because of a stiffening effect 15 due to the formation of rotational elastic conditions in the cables at the RHS ends. Notably, post-tensioned 16 thin-walled steel-girder-bridges draped with a deviator [43, 45, 46] almost never experience critical 17 buckling phenomena [Fig. 7(b)] as commonly happened in PC girder-bridges [17].



Fig. 12. Thin-walled member (Fig. 4). Effect of the post-tensioning force $(N_{0x,aver})$ on the fundamental frequency.

1 Fig. 13 shows the trend of *f*_{shear,1,*II*,FE of the thin-walled steel-girder-bridge with unbonded cables} 2 utilized in the numerical study presented by Bai-jian et al. [1]. Likewise, increasing the post-tensioning 3 force (N_{0x}) up to the critical buckling of the girder-cable system of 3,845 kN, $f_{\text{shear},1,U,FE}$ reduces the 4 negative slope of its softening branch according to the cables' eccentricity (e), i.e., from small (e/h=0.12)5 to medium (e/h = 0.28). The eccentric $N_{0x,ULS,0.12}$ and $N_{0x,pL0.12}$ are respectively equal to 433 and 669 kN 6 when e/h = 0.12. Vice versa, $N_{0x,ULS,0.28}$ and $N_{0x,pl,0.28}$ become 314 and 483 kN when e/h = 0.28 (Fig. 13). 7 Thin-walled member-cable systems with an eccentrically straight tendon have two sets of fundamental 8 vibrational modes, i.e., beam-dominated and tendon-dominated modes. Particularly, their dynamic 9 interaction depends not only on the magnitude of post-tensioning force (N_{0x}) but also on the location of 10 the tendon that induces an initial second-order curvature along the member. We can state that the vibration 11 beam mode follows the compression-softening theory when tendon's eccentricity (e) is zero, and it is 12 almost always dominant when the yielding stress under post-tensioning (N_{0x}) is not reached.





13

14 **7. Conclusions**

15 The effect of pre and post-tensioning forces on the beam dynamics is object of a significant discrepancy. 16 To solve this conflict, firstly, free transverse vibrations were induced to a post-tensioned thin-walled 17 steel-box-girder in larger scale and with eccentric straight cables. Secondly, to finally investigate the 18 validity of the compression-softening theory in prestressed beams, the experiments performed by Noble et al. [14] were analyzed. Within the limitations of this research and the results thoroughly validated using
 the bench-top experiment (Fig. 4), one proposed formula based on Timoshenko beam model [Eq. (4)],
 and two high-fidelity FE models including shear deformation, the following conclusions are reported:

41. The experimental dynamics were appropriately simulated using the FE model proposed by Jaiswal5[9] and that based on plate and shell elements. In thin-walled member–cable systems, both6modeling apply the effective post–tensioning force $(N_{0x,aver})$ in the form of initial tension in the7cables whilst neglect the corresponding second-order curvature. Accordingly, the fundamental8frequency $(f_{1,exp})$ of thin-walled steel members is not influenced by the beam's camber (initial9curvature) until the yielding stress (f_{yk}) under post–tensioning $(N_{0x,aver})$ is reached.

Conversely to what concluded by Noble et al. [14, 15], the dynamic effect of a compressive force
 and that of a post-tensioning phenomenologically differ only when the cables are in contact with
 the surrounding beam's section [Fig. 7(b)]. Depending on this characteristic, the dynamics of all
 forms of pre and post-tensioned members is described by compression-softening effect.

3. Similarly to what deduced for PC girder-bridges [7, 19, 20], the fundamental frequency ($f_{1,exp}$) as parameter for post-tensioning loss identification is made doubtful in thin-walled steel-girderbridges when the cables touch the surrounding section or are draped with a deviator [Fig. 8(b)].

17 4. When the beam's cambers (initial curvatures) are considered as deformed configurations within 18 the FE modeling, thin-walled member-cable systems have two sets of fundamental vibrational 19 modes (Figs. 12–13). By increasing the post–tensioning force ($N_{0x,aver}$) beyond the value causing 20 the yielding stress of the most extreme fibres of cross section (f_{yk}) , a stiffening effect occurs. This 21 consequence depends on the straight tendon's eccentricity (e): when it is small $(e/h = 0.12 \sim 0.23)$ 22 the softening effect is prevalent than the stiffening one and the vibration beam mode is dominant. 23 Vice versa, when the eccentricity is large $(e/h = 0.44 \sim 0.74)$ the stiffening effect prevails and the 24 vibration tendon mode is dominant (Figs. 12–13).

25

26 **CRediT authorship contribution statement**

M. Bonopera: Formulation and evolution of overarching research goals and aims, Funding acquisition,
Methodology, Numerical validation of theory using commercial software, Substantial contribution to

I	conception, Writing - original draft. KC. Chang: Methodology, Substantial contribution to acquisition
2	of data results, Substantial contribution to replication of results. N. Tullini: Funding acquisition, Ideas,
3	Methodology, Substantial contribution to acquisition of data results, Substantial contribution to
4	conception, Writing - review & editing.

6 **Declaration of competing interest**

7 The authors declare that they have no known competing financial interests or personal relationships that

8 could have appeared to influence the work reported in this paper.

1 1 C TZ

9

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- 18

19 Nomenclature

- 20 A : cross sectional area of the thin-walled Rectangular Hollow Section (RHS)
- 21 Af, A0, A1, A2, A3 : servo velocity seismometers
- 22 A_{tendon} : cross sectional area of straight tendon
- 23 A_w : cross sectional area of the web of a thin-walled RHS
- 24 *b* : internal width of a thin-walled RHS
- 25 *e* : eccentricity of straight tendon
- 26 *E* : Young's modulus of the thin-walled member
- 27 E_{tendon} : Young's modulus of straight tendon
- 28 $f_{1,I,FE,P\&S}$: first-order fundamental frequency based on FE plate/shell modeling
- 29 $f_{1,I,FE,P\&S,D}$: first-order fundamental frequency with a deviator based on FE plate/shell modeling
- 30 $f_{1,II,FE,P\&S}$: second-order fundamental frequency based on FE plate/shell modeling
- 31 $f_{1,II,FE,P\&S,1Link}$: second-order fundamental frequency with 1 rigid link based on FE plate/shell modeling
- 32 $f_{1,II,FE,P\&S,3Links}$: second-order fundamental frequency with 3 rigid links based on FE plate/shell modeling
- 33 $f_{1,II,FE,P\&S,D}$: second-order fundamental frequency with a deviator based on FE plate/shell modeling
- $f_{1,II,FE,P\&S,NoMasses}$: second-order fundamental frequency without masses at the RHS ends based on FE
- 35 plate/shell modeling
- 36 $f_{1,exp}$: experimental fundamental frequency
- 37 $f_{1,exp,max}$: experimental maximum fundamental frequency
- $f_{1,exp,min}$: experimental minimum fundamental frequency
- 39 $f_{\text{E-B},1,I}$: first-order fundamental frequency based on Euler-Bernoulli model

- $f_{\text{E-B},1,l,\text{FE}}$: first-order fundamental frequency based on FE model
- $f_{\text{E-B},1,I,\text{FE},W}$: first-order fundamental frequency with mass W based on FE model
- $f_{\text{E-B},1,II}$: second-order fundamental frequency based on Euler-Bernoulli model
- $f_{\text{E.-B.},1,II,\text{FE}}$: second-order fundamental frequency based on FE model
- $f_{\text{E-B},1,II,\text{FE},W}$: second-order fundamental frequency with mass W based on FE model
- $f_{\text{E-B},n,II}$: second-order *n*th natural frequency based on Euler-Bernoulli model
- $f_{\text{shear},1,I}$: first-order fundamental frequency based on shear model
- $f_{\text{shear},1,I,\text{FE}}$: first-order fundamental frequency based on FE model including shear deformation
- $f_{\text{shear},1,I,\text{FE},W}$: first-order fundamental frequency with mass W based on FE model including shear 10 deformation
- $f_{\text{shear},1,I,\text{FE},W,D}$: first-order fundamental frequency with mass *W* and deviator based on FE model including 12 shear deformation
- $f_{\text{shear},1,II}$: second-order fundamental frequency based on shear model
- $f_{\text{shear},1,II,\text{FE}}$: second-order fundamental frequency based on FE model including shear deformation
- $f_{\text{shear},1,II,\text{FE},W}$: second-order fundamental frequency with mass *W* based on FE model including shear 16 deformation
- $f_{\text{shear},1,II,\text{FE},W,D}$: second-order fundamental frequency with mass *W* and deviator based on FE model 18 including shear deformation
- $f_{\text{shear},n,l}$: first-order *n*th natural frequency based on shear model
- $f_{\text{shear},n,II}$: second-order *n*th natural frequency based on shear model
- F_d : release force of transverse vibration tests
- f_{yk} : yielding stress of the thin-walled member
- 23 g : gravitational acceleration
- G: shear modulus of the thin-walled member
- *h* : height of a thin-walled RHS
- *i* : measurement cross section along the thin-walled member
- *I* : cross sectional second moment of the area of the thin-walled member
- *k* : shear coefficient for a generic thin-walled RHS
- *L* : length of the thin-walled member
- 30 m: coefficient considering the shear stresses throughout the cross sectional area of a thin-walled RHS
- m_{beam} : thin-walled member's self-mass per unit length
- m_{tendon} : straight tendon's self-mass per unit length
- m_{tot} : thin-walled member's total self-mass per unit length corresponding to $m_{\text{beam}} + W/L$.
- $m_{tot,beam}$: thin-walled member's total self-mass
- M_{μ} : bending moments at the supports caused by sliding friction between steel-on-steel
- *n* : vibration beam mode
- N_{0x} : mean post-tensioning force
- N_{0x1} : applied post-tensioning force at the left RHS end
- N_{0x2} : applied post-tensioning force at the right RHS end
- $N_{0x,aver}$: mean applied post-tensioning force
- $N_{0x,aver,max}$: maximum mean applied post-tensioning force
- $N_{0x,pl,0.12}$: mean post-tensioning force causing the plasticity of cross section when e/h = 0.12
- $N_{0x,pl,0.23}$: mean post-tensioning force causing the plasticity of cross section when e/h = 0.23
- $N_{0x,pl,0.28}$: mean post-tensioning force causing the plasticity of cross section when e/h = 0.28
- $N_{0x,pl,0.74}$: mean post-tensioning force causing the plasticity of cross section when e/h = 0.74
- $N_{0x,ULS,0.12}$: mean post-tensioning force causing the yielding stress of the most extreme fibres of cross 47 section when e/h = 0.12
- $N_{0x,ULS,0.23}$: mean post-tensioning force causing the yielding stress of the most extreme fibres of cross 49 section when e/h = 0.23
- $N_{0x,ULS,0.28}$: mean post-tensioning force causing the yielding stress of the most extreme fibres of cross 51 section when e/h = 0.28
- $N_{0x,ULS,0.74}$: mean post-tensioning force causing the yielding stress of the most extreme fibres of cross
- 53 section when e/h = 0.74
- $N_{\rm crE,1}$: Euler buckling load

- 1 $N_{\text{crE},n}$: *n*th critical buckling load
- 2 $N_{\text{crE,shear,1}}$: first critical buckling load including shear deformation
- 3 $N_{\text{crE,shear},n}$: *n*th critical buckling load including shear deformation
- 4 *r* : radius of inertia of the thin-walled RHS
- 5 *t* : thickness of the web of a thin-walled RHS
- $6 t_{\rm I}$: thickness of the flanges of a thin-walled RHS
- 7 υ : Poisson's ratio of the thin-walled member
- 8 $v^{(0)}$: flexural shape of the fundamental frequency
- 9 W: mass of hydraulic oil jack
- 10 μ_{sliding} : frictional coefficient regarding the contact between steel-on-steel surfaces at the supports
- 11 ρ_{beam} : unit weight of the thin-walled member
- 12 ρ_{tendon} : unit weight of straight tendon
- 13 $\sigma_{tendon,max}$: maximum tensile strength reached in the straight tendon
- $\begin{array}{l} 14 \\ 15 \\ 16 \end{array} \quad \sigma_{uy} : \text{ yielding stress of straight tendon} \end{array}$
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